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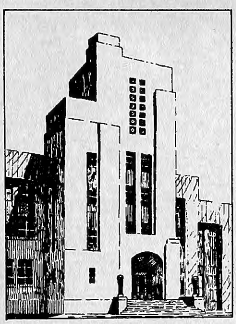
INVESTIGATION OF WAVE EFFECTS PRODUCED BY A THIN BODY - TMB MODEL 4125

by

Georg P. Weinblum, Janet J. Kendrick, and
M. Allison Todd



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NOTATION

a	One-half parallel length of model
B	Maximum beam of model
b	One-half maximum beam of model
C_r	Residual resistance coefficient
$C_w = \dfrac{R}{\frac{\rho}{2} Su^2}$	Calculated wave resistance coefficient
F	Froude number
f	Depth of immersion
g	Acceleration due to gravity
H	Draft
H_0	Struve's function
L	Total length of model
l	Parabolic length
M_1	$\int_0^1 \xi \sin \gamma \xi \, d \xi$
M_1'	$\int_0^1 \xi \cos \gamma \xi \, d \xi$
$p = \dfrac{2a}{L}$	Ratio of parallel length to total length
R	Resistance as calculated from Michell's integral
r	Wave resistance coefficient
S	Wetted surface
u	Velocity
Y_0	Bessel function of the second kind
Z_l	Surface elevation due to symmetrical disturbance
Z_w	Surface elevation due to wave system
α	Waterline coefficient
$\gamma_0 = \dfrac{1}{2F^2}$	
Δ	Displacement
δ	Block coefficient
ζ	Nondimensional vertical measurement

λ	Wave length
ξ	Nondimensional longitudinal measurement
ρ	Density
ϕ	Prismatic coefficient

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INTRODUCTION

An opportunity to check experimentally Michell's wave-resistance theory[1] arose when a friction plane or "thin body" was constructed for investigations of the frictional resistance of painted surfaces. In his wave-resistance theory, the only assumption made by Michell as to the form of the ship was that the inclination of the tangent plane at any point of its surface to the vertical median plane should be small.

Although the TMB model is quite narrow, it cannot be considered as a thin plank; thus, appreciable wave effects must be expected when the model is towed at medium and high Froude numbers.

The dimensions of the body are shown in Figure 1. It has a parallel middle body and parabolic ends with vertical sides. Apart from the rectangular form of the sections, this body forms an ideal Michell's ship as defined above due to its very low beam-length ratio [B/L = 0.0265] and beam-draft ratio [B/H = 0.183].

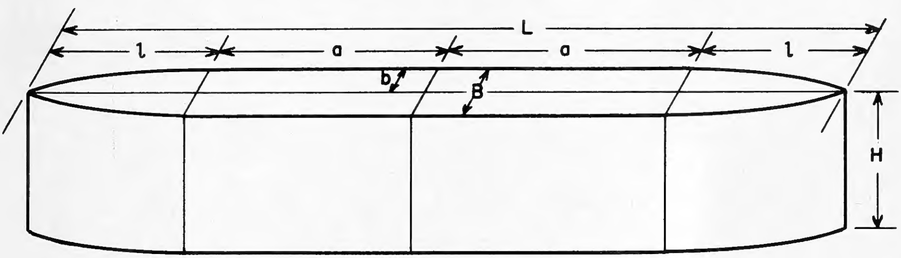

Figure 1 - Shape and Dimensions of Friction Body, TMB Model 4125

L = total length	= 252 inches	H = draft	= 36 inches
l = parabolic length	= 66 inches	a = half parallel length	= 60 inches
B = maximum beam	= 6.69 inches	p = 2a/L	= 0.4762
b = half beam	= 3.345 inches		

[1]References are listed on page 14.

It was decided to calculate the wave resistance experienced by the model and the wave profiles along the hull and to compare these theoretical values with the results of experiments.

Auxiliary resistance integrals, which simplify the work considerably, were not available at the time. Since, however, the draft-length ratio H/L is not covered by the systematic computations now being made at the Taylor Model Basin, an independent evaluation was needed in any case and justified the appreciable work involved.

RESISTANCE

The original form of Michell's resistance integral may be reduced to the dimensionless form[2]

$$R = \frac{8\rho g}{\pi} \frac{B^2 H^2}{L} \int_{\gamma_0}^{\infty} \left[I^{*2}(\gamma) + J^{*2}(\gamma) \right] \frac{\left(\frac{\gamma}{\gamma_0}\right)^2}{\sqrt{\left(\frac{\gamma}{\gamma_0}\right)^2 - 1}} d\gamma \qquad [1]$$

$$I^*(\gamma) = \int_0^1 \int_0^1 \frac{\partial \eta_a}{\partial \xi} e^{-2 \frac{H}{L} \frac{\gamma^2}{\gamma_0^2} \zeta} \cos \gamma \xi \, d\xi \, d\zeta \qquad [2]$$

$$J^*(\gamma) = \int_0^1 \int_0^1 \frac{\partial \eta_s}{\partial \xi} e^{-2 \frac{H}{L} \frac{\gamma^2}{\gamma_0^2} \zeta} \sin \gamma \xi \, d\xi \, d\zeta \qquad [3]$$

where $\xi = \frac{x}{l}$

$\gamma = \gamma_0 \lambda$

$\gamma_0 = \frac{gL}{2v^2} = \frac{1}{2F^2}$

η = waterline equation of hull

$\eta_s = \frac{1}{2}\left[\eta(x) + \eta(-x)\right]$

$\eta_a = \frac{1}{2}\left[\eta(x) - \eta(-x)\right]$

η_s, η_a represent the symmetrical and antisymmetrical parts of the hull with respect to the midship section, and

v = speed of advance.

Since for a symmetrical hull $I^*(\gamma) = 0$, Equation [1] reduces in this case to

$$R = \frac{8\rho g}{\pi}\,\frac{B^2H^2}{L}\int_{\gamma_0}^{\infty}\frac{\left(\frac{\gamma}{\gamma_0}\right)^2}{\sqrt{\left(\frac{\gamma}{\gamma_0}\right)^2-1}}\;J^{*2}(\gamma)\,d\gamma \qquad [4]$$

The intermediate function $J^*(\gamma)$ is given in this case by

$$J^*(\gamma) = -2E_0\left[\sin\gamma p M_1'\left\{\gamma(1-p)\right\} + \cos\gamma p M_1\left\{\gamma(1-p)\right\}\right] \qquad [5]$$

as can be seen by substituting the form of the waterline equation η_s in Equation [3].

Here

$$E_0 = \frac{1-e^{-\vartheta}}{\vartheta} \qquad \vartheta = \frac{2H\gamma^2}{L\gamma_0}$$

$$M_1' = \int_0^1 \xi\cos\gamma\xi\,d\xi$$

$$M_1 = \int_0^1 \xi\sin\gamma\xi\,d\xi$$

Graphs of E_0^2 over ϑ and of $\sin\gamma p M_1'\left\{\gamma(1-p)\right\} + \cos\gamma p M_1\left\{\gamma(1-p)\right\}$ over γ were plotted; from these the necessary values could be obtained.

The integral was evaluated at the singularity $\gamma/\gamma_0 = 1$ by integrating over a narrow range between $\gamma_0 + \gamma_0(1+\epsilon)$ where $\epsilon \leq 0.01$ using the formula

$$\int_{\gamma_0}^{\gamma_0(1+\epsilon)}\frac{\left(\frac{\gamma}{\gamma_0}\right)^2}{\sqrt{\left(\frac{\gamma}{\gamma_0}\right)^2-1}}\;J^{*2}(\gamma)\,d\gamma = \gamma_0\,J^{*2}(\gamma_0)\sqrt{2\epsilon}$$

The main part of the integral

$$\int_{\gamma_0(1+\epsilon)}^{\gamma_1}\frac{\left(\frac{\gamma}{\gamma_0}\right)^2}{\sqrt{\left(\frac{\gamma}{\gamma_0}\right)^2-1}}\;J^{*2}(\gamma)\,d\gamma$$

was evaluated from the graphs of the integrand by means of a planimeter. The remainder

$$\int_{\gamma_1}^{\infty}\frac{\left(\frac{\gamma}{\gamma_0}\right)^2}{\sqrt{\left(\frac{\gamma}{\gamma_0}\right)^2-1}}\;J^{*2}(\gamma)\,d\gamma$$

was calculated from expansion of the integrand.

The resistance was then calculated for two additional drafts, H = 20 inches and 10 inches, and at two Froude numbers for an infinite draft.

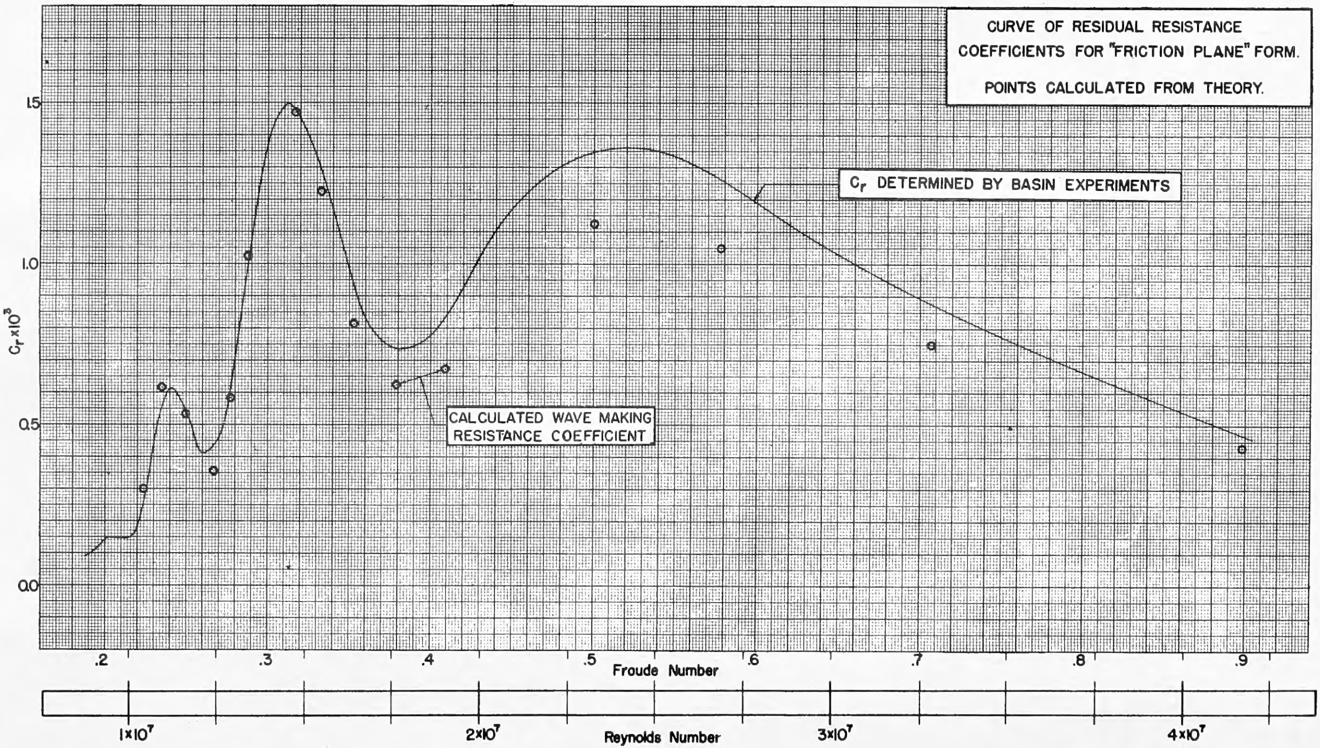

Figure 2 - Comparison of Calculated and Observed Wave Resistance of Model
(Reference 4)

Havelock has given an example showing how to correct resistance data obtained from friction body experiments for the wave resistance.[3] In the discussion of this paper, Weinblum has pointed out that these estimates may be much too low for moderate Froude numbers since they are based on a parabolic waterline while the actual form can be much fuller (see page 269, Reference 3).

The results of the computations are presented in various ways. Figure 2, reproduced from Reference 4, shows the comparison of the calculated wave resistance coefficient $C_w = \dfrac{R}{\frac{\rho}{2} S u^2}$ with the "residual resistance" coefficient C_r. Schoenherr's frictional resistance line was used to evaluate C_w.

The coefficients reach their maximum at moderate Froude numbers which is characteristic of very full forms. Figure 3 shows the calculated wave resistance coefficients $r = \dfrac{R}{\frac{8\rho g}{\pi} \frac{B^2 H^2}{L}}$ plotted against $\gamma_0 = \dfrac{1}{2F^2}$ for three depth-length ratios.

Figure 4 shows the wave resistance coefficients of a parabolic form (for which $p = 0$) with the same proportions as the friction body. It can be seen that for this form, the coefficient of resistance is much lower than that of the fuller form except in the region of the large hump.

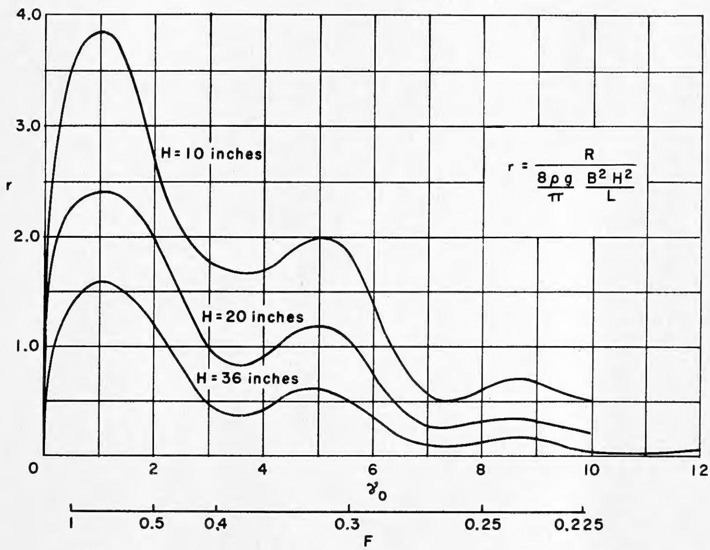

Figure 3 - Wave Resistance Coefficients r Plotted Against Speed for Three Different Drafts

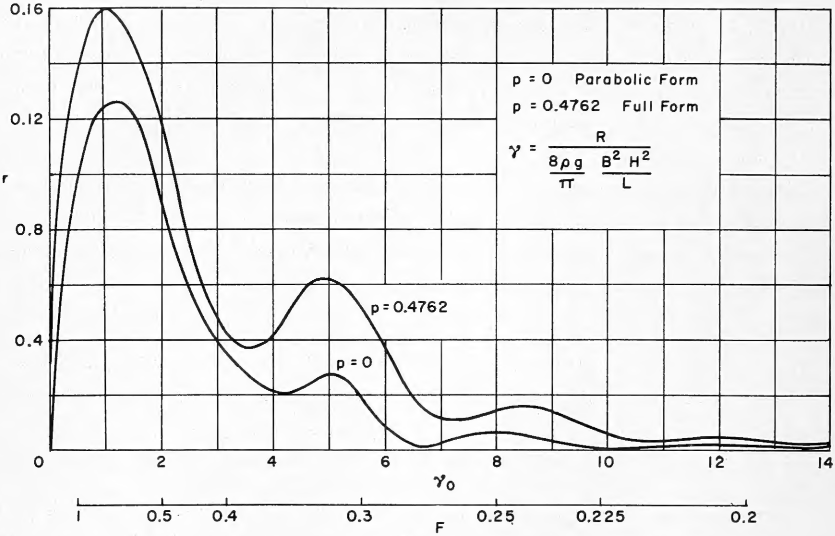

Figure 4 - Comparison of Wave Resistance Coefficients of Parabolic and Full Forms

To obtain a better basis of comparison between theory and experiment, it was suggested that the influence of the sharp corners on the resistance should be eliminated by towing the model at a shallower draft $H = 20$ inches. Then the measured and calculated differences in resistance for the two drafts could have been calculated. However, it was not feasible to tow the model at a shallower draft at the time the tests were run.

WAVE PROFILES

The work done on the calculation of the wave profiles of the friction body was based upon information given by W.C.S. Wigley.[5] His investigations were performed upon a body having a short parallel middle body, 1/16 of the total length, and parabolic ends.

Since the subject of wave profiles has not been treated as frequently as that of wave resistance, the computations are presented here in detail.

Two formulas given by Wigley in dimensional form were used in the calculations of the wave profiles. The first gave the surface elevation due to wave making and the second the symmetrical disturbance.

In dimensionless coefficients these become

$$\frac{\pi\gamma(1-p)Z_w}{8b} = P_0\left\{\gamma\xi\right\} + P_0\left\{\gamma(\xi-2)\right\} - \frac{1}{\gamma(1-p)}\Big[P_0^{-1}\left\{\gamma\xi\right\} - P_0^{-1}\left\{\gamma(\xi-1+p)\right\} - P_0^{-1}\left\{\gamma(\xi-2)\right\} + P_0^{-1}\left\{\gamma(\xi-1-p)\right\}\Big] \qquad [6]$$

and

$$\frac{\pi\gamma(1-p)Z_l}{2b} = -\Big[Q_0\{\gamma\xi\}+Q_0\{\gamma(\xi-2)\} - \frac{1}{\gamma(1-p)}\Big(Q_1\{\gamma\xi\} \qquad [7]$$

$$- Q_1\{\gamma(\xi-1+p)\}+Q_1\{\gamma(\xi-1-p)\}-Q_1\{\gamma(\xi-2)\}\Big)\Big]$$

where Z_l is the surface elevation due to non-wave portion of disturbance caused by motion of the form,* and Z_w is the surface elevation due to wave system.

The nondimensional longitudinal distance ξ is positive when measured in the astern direction. The basic functions P and Q are defined as follows:[7, 8]

$$P_0(u) = \int_0^{\frac{\pi}{2}} \sin(u\sec\phi)\,d\phi$$

$$P_0^{-1}(u) = 1+P_1(u) = 1-\int_0^{\frac{\pi}{2}} \cos\phi\cos(u\sec\phi)\,d\phi$$

In the present calculations, the values of the P functions were obtained from a graph prepared by Professor Lunde.[7]

$$Q_0(u) = \frac{\pi}{2}\int_0^u \Big\{H_0(t)-Y_0(t)\Big\}\,dt$$

where $H_0(t)$ is Struve's function and $Y_0(t)$ is the Bessel function of the second kind. The values of $Q_0(u)$ may be calculated for values of $t \le 16$ using the tables of $H_0(t)$ and $Y_0(t)$ given by Watson.[9] For values of $t > 16$ (for which the values of $H_0(t)$ and $Y_0(t)$ are not given), the following method was used:

When t is large (in this case for $t > 16$)

$$H_0(t) = Y_0(t) + \frac{1}{\pi}\sum_{m=0}^{m} \frac{\Gamma\left(m+\frac{1}{2}\right)}{\Gamma\left(\frac{1}{2}-m\right)\left(\frac{1}{2}t\right)^{2m+1}}$$

Therefore

$$Q_0(t) - Q_0(16) = \frac{1}{2}\int_{16}^{u}\sum_{m=0}^{m} \frac{\Gamma\left(m+\frac{1}{2}\right)}{\Gamma\left(\frac{1}{2}-m\right)\left(\frac{1}{2}t\right)^{2m+1}}\,dt$$

*"The function Q_0 does not oscillate but is monotonic, and the terms in Q_0 represent a symmetrical disturbance of the surface in the neighborhood of the form which dies away quickly both fore and aft of the form. It absorbs no energy owing to its symmetry and therefore does not affect the resistance."[6]

TABLE 1

Values for $Q_0(u) = \frac{\pi}{2}\int_0^u [H_0(t) - Y_0(t)]\, dt$

u	$Q_0(u)$	u	$Q_0(u)$	u	$Q_0(u)$
0	0	9.5	3.538	20.5	4.303
0.2	0.563	10.0	3.589	21.0	4.327
0.4	0.880	10.5	3.637	21.5	4.350
0.6	1.118	11.0	3.681	22.0	4.373
0.8	1.311	11.5	3.727	22.5	4.396
1.0	1.474	12.0	3.770	23.0	4.418
1.5	1.801	12.5	3.810	23.5	4.439
2.0	2.051	13.0	3.849	24.0	4.460
2.5	2.244	13.5	3.887	24.5	4.481
3.0	2.420	14.0	3.923	25.0	4.501
3.5	2.565	14.5	3.958	25.5	4.521
4.0	2.693	15.0	3.991	26.0	4.540
4.5	2.806	15.5	4.024	26.5	4.559
5.0	2.908	16.0	4.056	27.0	4.578
5.5	3.000	16.5	4.087	27.5	4.596
6.0	3.085	17.0	4.116	28.0	4.614
6.5	3.164	17.5	4.145	28.5	4.632
7.0	3.237	18.0	4.173	29.0	4.649
7.5	3.333	18.5	4.201	29.5	4.666
8.0	3.368	19.0	4.227	30.0	4.683
8.5	3.428	19.5	4.253		
9.0	3.485	20.0	4.278		

TABLE 2

Values for $Q_1(u) = \int_0^u Q_0(t)\,dt$

u	$Q_1(u)$	u	$Q_1(u)$	u	$Q_1(u)$
0	0	9.5	24.988	20.5	68.636
0.2	0.065	10.0	26.775	21.0	70.803
0.4	0.209	10.5	28.577	21.5	72.963
0.6	0.412	11.0	30.411	22.0	75.153
0.8	0.653	11.5	32.258	22.5	77.336
1.0	0.933	12.0	34.138	23.0	79.549
1.5	1.755	12.5	36.028	23.5	81.753
2.0	2.718	13.0	37.947	24.0	83.988
2.5	3.796	13.5	39.877	24.5	86.213
3.0	4.959	14.0	41.834	25.0	88.468
3.5	6.211	14.5	43.799	25.5	90.714
4.0	7.521	15.0	45.791	26.0	92.989
4.5	8.902	15.5	47.791	26.5	95.254
5.0	10.325	16.0	49.815	27.0	97.548
5.5	11.808	16.5	51.846	27.5	99.832
6.0	13.324	17.0	53.902	28.0	102.144
6.5	14.892	17.5	55.958	28.5	104.446
7.0	16.487	18.0	58.047	29.0	106.776
7.5	18.132	18.5	60.131	29.5	109.095
8.0	19.810	19.0	62.247	30.0	111.442
8.5	21.504	19.5	64.358		
9.0	23.237	20.0	66.500		

Evaluating to the first two terms only

$$Q_0(t) - Q_0(16) = \frac{1}{2}\int_{16}^{u}\left[\frac{\sqrt{\pi}}{\sqrt{\pi}\,\frac{t}{2}} + \frac{\frac{1}{2}\sqrt{\pi}}{-2\sqrt{\pi}\,\frac{t^3}{8}}\right] dt$$

$$= \frac{1}{2}\int_{16}^{u}\left(\frac{2}{t} - \frac{2}{t^3}\right) dt \qquad [8]$$

$$= \log t + \frac{1}{2t^2} - \log 16 - \frac{1}{512}$$

$Q_0(u)$ was evaluated for $16 < t \leq 30$ using Equation [8]. A tabulation of these values is given in Table 1.

The values of $Q_1(u) = \int_0^u Q_0(t)\,dt$ were obtained from these by integration; see Table 2.

The P-functions are not defined for negative values of t. $P(-|t|) = 0$ was used. Thus, it can be seen from Equation [6] for Z_w that for $0 \leq \xi \leq 1 - p$ only the terms $P_0(\gamma\xi)$ and $P_0^{-1}(\gamma\xi)$ are defined. For $1 - p \leq \xi \leq 1 + p$ the term $P_0^{-1}\{\gamma(\xi - \overline{1-p})\}$ is defined. For $1 + p \leq \xi \leq 2$ the term $P_0^{-1}\{\gamma(\xi - \overline{1+p})\}$ is also defined and for $\xi \geq 2$ all are defined. It would appear then that the first two terms mentioned would give the ordinates of the bow wave system, the next the ordinates for the waves proceeding from the foreshoulder, the fourth the ordinates of the waves proceeding from the aftershoulder, and the last two the ordinates of the stern wave system.

In the example given in the present paper (Figure 6), these four waves are plotted separately to show the components of the total wave system contributed by each of the terms considered above.

The Q-functions appearing in the equation for the symmetrical disturbance[7] are defined for negative values of t as:

$$Q_0(-t) = Q_0(t) \qquad \text{and} \qquad Q_1(-t) = -Q_1(t)$$

Thus all the terms of Equation [7] are defined for all values of ξ.

The evaluation of Equations [6] and [7] give the wave profiles in terms of dimensionless coefficients containing Z_w and Z_l. The multiplication of these values by the constants $\frac{8b}{\pi\gamma(1-p)}$ and $\frac{2b}{\pi\gamma(1-p)}$ respectively, transforms them into the dimensions of b. In Figure 7 the calculated and observed profiles are plotted to an inch scale. In Figure 6, however, the components are plotted in dimensionless form.

The comparison was made as outlined above for five different speeds. At the lower speeds (4.5 and 5 knots) the agreement between the observed and calculated profiles is good; for higher speeds the agreement of the crests is good but the troughs of the calculated profiles are deeper than those observed (Figure 7). This is at least partly accounted for by the assumption of infinite draft made throughout the calculation. The influence of a finite draft will be more marked at higher speeds. This will be evident from a consideration of the

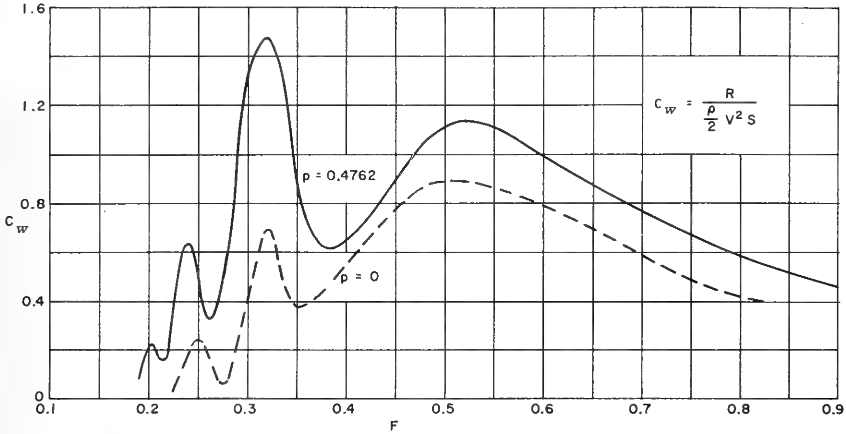

Figure 5 - Wave Resistance Coefficient C_w Plotted Against Froude Number for Parabolic and Full Forms

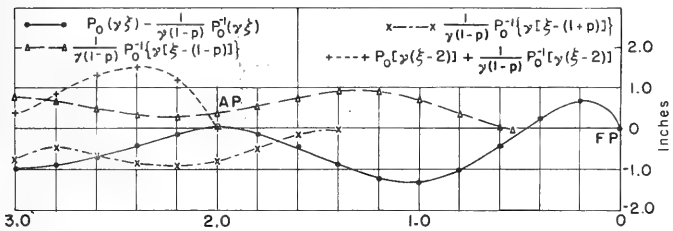

Figure 6a - Main Surface Elevation

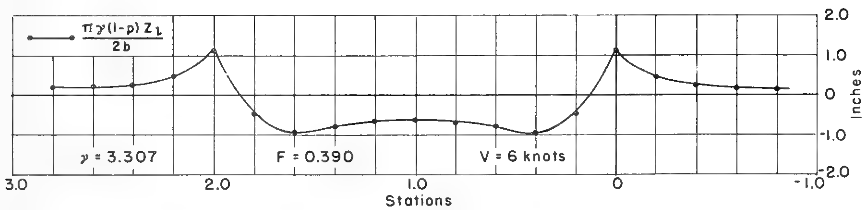

Figure 6b - Symmetrical Disturbance

Figure 6 - Wave Profiles for Speed of Advance of 6 Knots Plotted in Dimensionless Coefficients and Showing Components

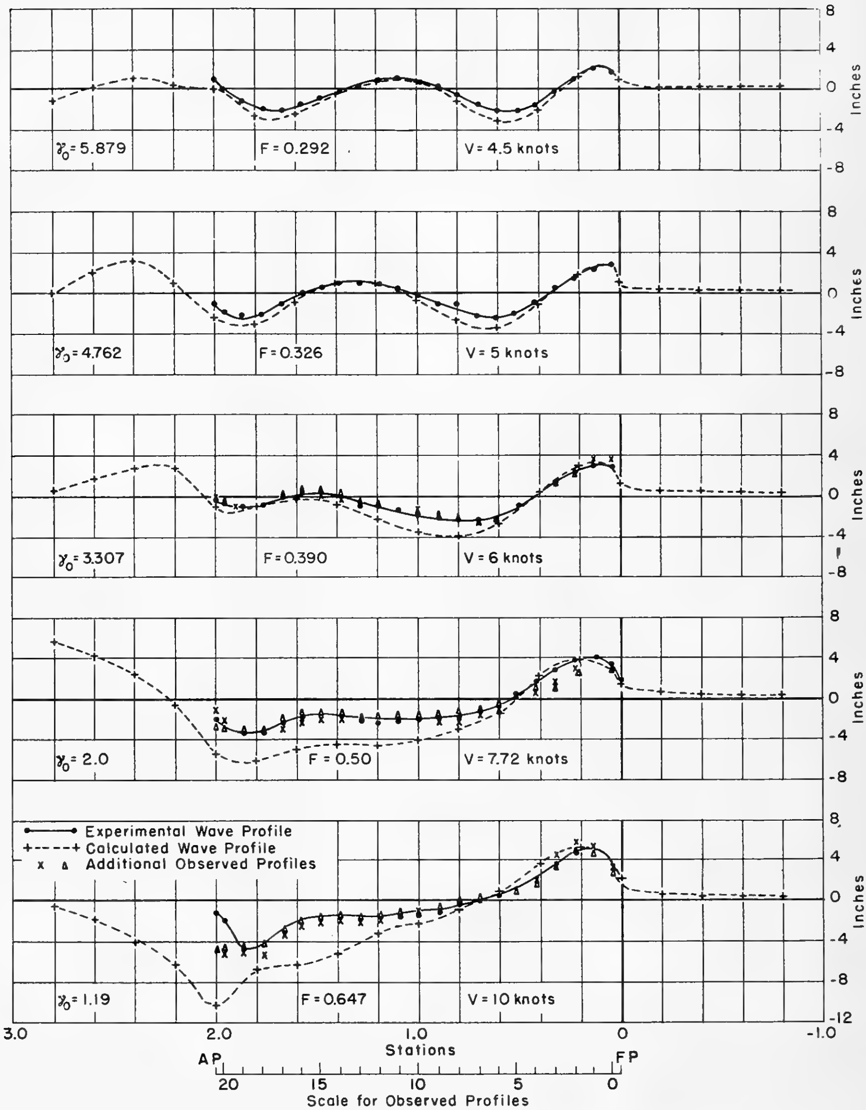

Figure 7 - Comparison of Calculated and Measured Wave Profiles for Five Different Speeds of Advance

exponential term relating the below-surface disturbing pressures to the wave length at the surface and the depth of immersion. These forces vary approximately as $e^{-\frac{2\pi f}{\lambda}}$ where f is the depth of immersion and λ is the wave length. Thus as λ increases with increasing speed, $e^{-\frac{2\pi f}{\lambda}}$ also increases. The effect upon the water surface elevation of the finite draft of the model will, therefore, be greater at higher speeds than at low, and better agreement than that actually obtained cannot be expected.

It was found that measuring the wave profile at the bow sometimes presents serious difficulties because of the steep slope of the wave in a transverse plane. Photographs of wave contours along the ship may therefore prove to be rather unreliable.

CONCLUSION

The qualitative and quantitative agreements between the calculated wave resistance coefficient C_w and the residual resistance coefficient C_r (as shown in Figure 2) are very satisfactory. In particular, prediction of the "third" hump* was possible in the resistance curve of Figure 3 at $F = 0.24$ which at first escaped the attention of the experimenters.

Because of this good agreement, the following deductions from the computations can be relied upon without further checks:

It is well known that because of the linear character of Michell's analysis the relation $R \sim B^2$ holds.

It can also be demonstrated that at very large Froude numbers ($\gamma \to 0$) the integral r becomes independent of H/L provided this ratio is finite. Thus the three curves on Figure 3 must finally coincide and $R \sim B^2H^2$, i.e., the wave resistance in this range is proportional also to the square of the depth H. However, it can be seen from the figures that this asymptotic relation does not hold for high-speed displacement ships. For example, at a destroyer speed of $F = 0.6$, $\gamma = 1.5$. In this case R is proportional to $H^{1.5}$ only; at $F = 0.25$, $R \sim H^2$ where r is already less than unity.

The studies of the method of formation of wave systems described in this report explain to some extent the characteristic form of the wave-resistance curve.

It can readily be seen that with increasing speed and therefore with increasing wave length, the crests of the bow wave system will alternately reinforce and dampen those of the stern and shoulder wave systems. Similarly, combinations of these systems will change with changing speed. These reinforcing and damping effects of the wave systems account for the

*The hump in the resistance coefficient curve occurring at $F \cong 0.5$ is here denoted as the "first" hump, at $F \cong 0.3$ as the "second" hump, etc. (see Figure 3), contrary to the custom in naval architecture by which the hump at the highest speed is called the last hump. This change appeared to be necessary since from a mathematical viewpoint there are an infinite number of humps between $0.5 \geq F \geq 0$ (see also Reference 2, page 23).

peaks and hollows seen in the curve of wave resistance plotted against speed. However, after a certain speed is reached, even the second wave of the bow wave system will be astern of the ship, and there can be no further reinforcement of the other systems. Above this speed, therefore, the wave resistance curve will have no further peaks but will show a steady fall with further increase in speed.

ACKNOWLEDGMENT

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